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USACE / NAVFAC / AFCEA / NASA UFGS-43 21 13.00 40 (April 2006)  
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Preparing Activity: NASA Superseding  
NASA-15135S (December 2005)

## UNIFIED FACILITIES GUIDE SPECIFICATIONS

References are NOT in agreement with UMRL dated 01 April 2006

Revised throughout - changes not indicated by CHG tags

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### SECTION 43 21 13.00 40

#### CENTRIFUGAL PUMPS

04/06

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NOTE: Delete, revise, or add to the text in this section to cover project requirements. Notes are for designer information and will not appear in the final project specification.

This section covers centrifugal pumps.

Schedule project chilled-water, dual-temperature, and condenser water pumps by sequential numbering with the letter P prefix. For example: P-1, P-2, P-3, etc. Schedule class of pump: e.g., Class ADS, as specified herein.

Under the heading "Special Requirements," specify any detail, deviation, or waiving of requirements, which is pertinent only to specific pumps.

The following parameters cover a wide spectrum of specification criteria to suit project conditions. Selections shall be carefully checked to ensure equivalence and compliance by at least three manufacturers.

Motors are covered in Section 26 18 39.00 40 MOTORS.

Comments and suggestions on this guide specification are welcome and should be directed to the technical proponent of the specification. A listing of technical proponents, including their organization designation and telephone number, is on the Internet.

Recommended changes to a UFGS should be submitted as a Criteria Change Request (CCR).

Use of electronic communication is encouraged.

Brackets are used in the text to indicate designer choices or locations where text must be supplied by the designer.

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## PART 1 GENERAL

### 1.1 REFERENCES

\*\*\*\*\*

NOTE: This paragraph is used to list the publications cited in the text of the guide specification. The publications are referred to in the text by basic designation only and listed in this paragraph by organization, designation, date, and title.

Use the Reference Wizard's Check Reference feature when you add a RID outside of the Section's Reference Article to automatically place the reference in the Reference Article. Also use the Reference Wizard's Check Reference feature to update the issue dates.

References not used in the text will automatically be deleted from this section of the project specification when you choose to reconcile references in the publish print process.

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The publications listed below form a part of this specification to the extent referenced. The publications are referred to within the text by the basic designation only.

#### AMERICAN BEARING MANUFACTURERS ASSOCIATION (ABMA)

ANSI/ABMA 11 (1990; R 1999) Load Ratings and Fatigue Life for Roller Bearings

ANSI/ABMA 9 (1990; R 2000) Load Ratings and Fatigue Life for Ball Bearings

#### ASME INTERNATIONAL (ASME)

ASME B16.1 (1998) Cast Iron Pipe Flanges and Flanged Fittings Classes 25, 125, and 250

#### HYDRAULIC INSTITUTE (HI)

HI SCRRP (1994) Standards for Centrifugal, Rotary and Reciprocating Pumps

#### INTERNATIONAL ORGANIZATION FOR STANDARDIZATION (ISO)

ISO 1940-1 (2003) Mechanical Vibration - Balance Quality Requirements for Rotors in a Constant (Rigid) State - Part 1: Specification and Verification of Balance Tolerance - International Restrictions

ISO 2858 (1975) End Suction Centrifugal Pump (Rating 16 Bar) Designation Nominal Duty

Point and Dimensions - International  
Restrictions

ISO 5199

(2002) Technical Specifications for  
Centrifugal Pumps, Class II

ISO 7005-2

(1988) Metallic Flanges Part 2: Cast Iron  
Flanges

MANUFACTURERS STANDARDIZATION SOCIETY OF THE VALVE AND FITTINGS  
INDUSTRY (MSS)

MSS SP-51

(2003) Class 150 LW Corrosion Resistant  
Cast Flanges and Flanged Fittings

MSS SP-86

(2002) Guidelines for Metric Data in  
Standards for Valves, Flanges, Fittings  
and Actuators

NATIONAL ELECTRICAL MANUFACTURERS ASSOCIATION (NEMA)

ANSI/NEMA MG 1

(2003) Standard for Motors and Generators

1.2 DESIGN REQUIREMENTS

[Pumps shall be designed using hydraulic criteria based upon actual model developmental test data. Manufacturer shall certify that pumps have been hydraulically tested at the factory.]

Pumps shall be selected at a point within the maximum efficiency for a given impeller casing combination. Deviations within 3 percent of maximum efficiency are permissible, provided the lesser efficiency is not less than the scheduled efficiency.

Pumps having impeller diameters larger than 90 percent of the published maximum diameter of the casing or less than 15 percent larger than the published minimum diameter of the casing will be rejected.

Acceptable maximum impeller diameter calculations shall not be based on percentage of impeller diameter range for a given casing. Shop drawings will be approved only if complete performance curves for all impeller sizes for a given casing are included in the submittal.

[Where parallel-pump operation is indicated, pumps selected shall have characteristics specifically suitable for the service without unstable operation.]

[When Net Positive Suction Head (NPSH) calculations are made, inlet-storage head shall be considered at empty point and the centerline pump location shall include not less than a 150 millimeter 6-inch concrete base.]

[Pumps shall be suitable for operation at indicated temperature without vapor binding and without cavitation under any system operating condition. The only acceptable means of rectification of cavitation shall be replacement of entire pump assembly.]

Available Net Positive Suction Head (NPSH) shall exceed required NPSH by not less than 0.46 meter 1-1/2 feet.

Pumps of the same duty condition, classification, and accessories, or with specified accessory deviation, shall be identical and the product of one manufacturing source.

Pumps from more than one manufacturing source shall be provided only when a single manufacturing source is unable to meet all specification requirements.

### 1.3 SUBMITTALS

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NOTE: Review Submittal Description (SD) definitions in Section 01 33 00 SUBMITTAL PROCEDURES and edit the following list to reflect only the submittals required for the project. Submittals should be kept to the minimum required for adequate quality control.

A "G" following a submittal item indicates that the submittal requires Government approval. Some submittals are already marked with a "G". Only delete an existing "G" if the submittal item is not complex and can be reviewed through the Contractor's Quality Control system. Only add a "G" if the submittal is sufficiently important or complex in context of the project.

For submittals requiring Government approval on Army projects, a code of up to three characters within the submittal tags may be used following the "G" designation to indicate the approving authority. Codes for Army projects using the Resident Management System (RMS) are: "AE" for Architect-Engineer; "DO" for District Office (Engineering Division or other organization in the District Office); "AO" for Area Office; "RO" for Resident Office; and "PO" for Project Office. Codes following the "G" typically are not used for Navy, Air Force, and NASA projects.

Choose the first bracketed item for Navy, Air Force and NASA projects, or choose the second bracketed item for Army projects.

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Government approval is required for submittals with a "G" designation; submittals not having a "G" designation are [for Contractor Quality Control approval.] [for information only. When used, a designation following the "G" designation identifies the office that will review the submittal for the Government.] Submit the following in accordance with Section 01 33 00 SUBMITTAL PROCEDURES:

#### SD-02 Shop Drawings

Installation drawings for centrifugal pumps shall be submitted in accordance with Part 3, "Execution," of this section.

#### SD-03 Product Data

The following shall be submitted for centrifugal pumps in

accordance with paragraph entitled, "General Requirements," of this section.

Equipment and Performance Data  
Equipment Foundation Data

#### SD-05 Design Data

Design Analysis and Calculations shall be submitted for centrifugal pumps in accordance with paragraph entitled, "General Requirements," of this section.

#### SD-06 Test Reports

Test reports for pumps shall be submitted on the following tests:

Hydraulic Tests  
Efficiency Tests  
Vibration Tests  
Output Efficiency  
Surface Hardness Tests  
Deflection Tests

#### SD-07 Certificates

Certificates shall be submitted for the following items showing conformance with the referenced standards contained in this section.

Base-Mounted Centrifugal Pumps  
Line-Mounted Pumps  
Accessories

### 1.4 GENERAL REQUIREMENTS

\*\*\*\*\*  
NOTE: If Section 23 00 00.00 40 GENERAL MECHANICAL PROVISIONS is not included in the project specification, applicable requirements therefrom should be inserted and the following paragraph deleted. If Section 23 05 48.00 40 VIBRATION ISOLATION FOR AIR CONDITIONING EQUIPMENT is not included in the project specification, applicable requirements therefrom should be inserted and the second paragraph deleted. If Section 26 18 39.00 40 MOTORS is not included in the project specification, applicable requirements therefrom should be inserted and the third paragraph deleted.  
\*\*\*\*\*

[Section 23 00 00.00 40 GENERAL MECHANICAL PROVISIONS applies to work specified in this section.]

[Section 23 05 48.00 40 VIBRATION ISOLATION FOR AIR CONDITIONING EQUIPMENT applies to work specified in this section.]

[Section 26 18 39.00 40 MOTORS applies to this section.]

Design Analysis and Calculations shall show NPSH (net positive suction

head) calculations for centrifugal pumps.

Equipment and Performance Data consisting of pump curves (liter gallons per minute versus total head in millimeter feet per rpm) shall be provided for each type centrifugal pump.

Equipment Foundation Data shall be submitted for centrifugal pumps.

#### 1.4.1 Factory Tests

The following tests shall be submitted from the manufacturer prior to shipping the pump from the factory, hydraulic tests, efficiency tests, vibration tests, output efficiency tests, surface hardness tests, and shaft deflection tests.

### PART 2 PRODUCTS

\*\*\*\*\*  
NOTE: Paragraph entitled, "General Pump Requirements," applies to engineered quality pumps. See paragraph entitled, "Base-Mounted Centrifugal Pumps," for building-trades quality base-mounted pumps.  
\*\*\*\*\*

\*\*\*\*\*  
NOTE: Pump and motor balance shall conform to ISO 1940-1 - (1986) Balance Quality Requirements of Rigid Rotors - Determination of Permissible Residual Unbalance unless otherwise noted. Motor vibration levels shall conform to NEMA Specification MG-1, Motors and Generators, Part 7 unless otherwise noted.  
\*\*\*\*\*

#### 2.1 GENERAL PUMP REQUIREMENTS

Certificates for pumps and Accessories shall show conformance with the referenced standards contained in this section.

This specification includes design, construction, installation, and performance features of centrifugal water pumps. Pumps provided shall conform to ISO 5199 and ISO 2858 HI SCRRP standards for centrifugal pumps, and to requirements specified herein.

##### 2.1.1 Classification

Class ADS: Axially (horizontally) split-case, single-stage, double-suction, single- or double-volute centrifugal type

\*\*\*\*\*  
NOTE: Limit all end-suction pumps to 1,000 gpm at 100 psi 4.0 cubic meter per minute at 690 kilopascal.  
\*\*\*\*\*

Class FES: Radially (vertically) split-case, single-stage, frame-mounted distance piece or back pullout end suction, single- or double-volute centrifugal type

Class CES: Radially (vertically) split-case, single-stage, close-coupled,



distance piece end-suction, single- or double-volute centrifugal type

#### 2.1.1.2 Casing

Pump casings shall be bronze-fitted, seasoned cast iron with a design working pressure of not less than 1275 kilopascal at 38 degrees C 185 pounds per square inch gage (psig) at 100 degrees F. Casings shall be single or double volute with flanged piping connections conforming to ASME B16.1, MSS SP-51, MSS SP-86 and ISO 7005-2, Class 125 psi. Direction of shaft rotation shall be conspicuously indicated. Casing shall have tapped openings for air venting, priming, draining, and suction and discharge gages. A brass or bronze umbrella or vent cock shall be furnished for venting except where automatic air vents are indicated. Drain openings in the volute, intake, or other passages capable of retaining trapped water shall be located in the low point of such passages.

Casing construction shall be such that packing seals may be substituted in the field for mechanical seals without machining.

\*\*\*\*\*  
NOTE: If a paragraph is applicable to a certain  
project pump only, rewrite and identify project pump  
number.  
\*\*\*\*\*

Packing-box depth shall accommodate [six rings of square packing, lantern ring, and throttling bushing.] [not less than five rings of square packing, lantern ring, and throttling bushing.] [not less than four rings of square packing, lantern ring, and throttling bushing.]

#### 2.1.1.3 Distance Piece

A suction end distance piece shall be provided with each end suction pump as a part of the pump or piping system.

#### 2.1.1.4 Impellers

Impellers shall be enclosed cast bronze or corrosion-resistant steel, machined and polished. Waterways shall be machine- or hand-finished. Impellers shall meet maximum and minimum diameter requirements.

#### 2.1.1.5 Balancing

\*\*\*\*\*  
NOTE: Balance is the process of improving the mass  
distribution of the pump components in order to  
minimize damaging centrifugal forces. No component  
can be perfectly balanced. there will always be some  
remaining unbalance. The minimum recommended balance  
grade is ISO grade 2.5. Select an ISO grade or  
insert a standard.  
\*\*\*\*\*

Pump impeller assemblies shall be statically and dynamically balanced to ISO 1940-1-1986, [G6.3] [G2.5] [G1.0] [ ]. Correction planes needed for additional mass weight for balancing shall be determined by using a calibrated and certified balancing machine capable of identifying the magnitude and angular position of any unbalance of the impeller.

#### 2.1.1.6 Wearing Rings

Wearing rings shall be dissimilar bronze composition for nongalling service. Wearing rings shall be provided in every pump case and on all impellers larger than 175 millimeter 7 inches in diameter.

#### 2.1.1.7 Shaft

[Shafts for convertible packed or mechanical seal service shall be solid sleeveless AISI 400 series corrosion-resistant steel hardened to 425 Brinell in packing area, or sleeved type with AISI 300 series corrosion-resistant steel shaft and AISI 400 series corrosion-resistant steel sleeves hardened to 425 Brinell.]

[Shafts for packed seal service shall be sleeved and all materials shall be AISI Type 304 corrosion-resistant steel. Surfaces in packing area shall be plasma-spray-coated alumina ceramic with a coefficient of expansion compatible with the substrate. Surface hardness shall be 900 Brinell.]

[Shafts for mechanical-seal service shall be solid or sleeved and all materials shall be AISI Type 304 or 316 corrosion-resistant steel.]

[Press-fitted sleeves are not acceptable.]

\*\*\*\*\*  
NOTE: Select the following paragraph only when  
Class CES pumps are specified.  
\*\*\*\*\*

Motor shafts of close-coupled pumps shall be manufacturer's standard AISI 18-8 corrosion-resistant steel and finish.

Surfaces shall have a 406 nanometer 16-microinch surface finish where packing is specified or where a pump must be convertible to packing seals from mechanical seals. Pump shafts to be sealed by mechanical seals only shall have a 815 nanometer 32-microinch surface finish, or better.

[End-suction pump shafts shall have an impeller nut that completely encloses shaft end threads and seals tight to the impeller.]

\*\*\*\*\*  
NOTE: If a service requires extended operation at  
or near shutoff head or far out on the curve, this  
operating condition must be specified. Check  
permissible shaft deflection for possible revision.  
  
Specify manual or automatic bypass system if  
necessary.  
\*\*\*\*\*

Shaft construction shall be substantial to prevent seal or bearing failure due to vibration. Shaft vibration at pump-seal face shall conform to the paragraph entitled, "Pump Acceptance," under shutoff-head operating conditions. Flow from DN6 1/4-inch iron pipe size (ips) pipe shall be provided during testing.

Shaft shall be equipped with bronze or nylon water slingers at each bearing and shall be sealed at the casing interface with a bronze throttling bushing.

### 2.1.8 Packing Seals

Packing shall be soft woven non-asbestos material with not less than 25 percent by weight of tetrafluoroethylene resin. Pump shall be shipped to the site without the packing inserted and shall be packed on site in the presence of the pump or packing manufacturer's representative. At no time during startup or run-in shall the gland drip less water than 80 drops per minute. After a minimum of 40 operating hours and upon permission of the Contracting Officer, leakage rate may be reduced to 50 drops per minute or to the rate recommended by packing manufacturer.

Gland shall be split-bronze type with AISI 18-8 corrosion-resistant steel eyebolts and pins or studs. Hex-nuts shall be bronze or nongalling corrosion-resistant steel.

Stuffing boxes exposed to below atmospheric pressure at any operating condition, including starting, shall be provided with a water seal. Water seal shall consist of nonferrous lantern ring or a seal cage and required connections to the pump case.

### 2.1.9 Mechanical Seals

\*\*\*\*\*  
NOTE: Where pump duty conditions include severe on/off or extreme of either end-of-curve operation, pump should be packed only, or one packed and one mechanically sealed pump should be specified with the latter pump convertible to packing.  
\*\*\*\*\*

Mechanical seals shall be balanced or unbalanced, as necessary to conform to specified service requirements. Mechanical seals shall be constructed in a manner and of materials particularly suitable for the temperature service range and chemical analysis of water being pumped.

Cooling-water characteristics for seal construction purposes are as follows: makeup total dissolved solids of 200 parts per million (ppm) cycled up to five times, containing not more than 600 ppm of hexavalent chromate, and pH not less than 6.0.

Seal construction shall not require external source cooling for pumped-fluid service temperatures up to 121 degrees C 250 degrees F.

Seal pressure rating shall be suitable for maximum system hydraulic conditions.

Materials of construction shall include AISI 300 series corrosion-resistant steel, solid tungsten-carbide rotating-seal face, and Buna-N vinylidene-fluoride-hexafluoropropylene, EPT, or tetrafluoroethylene seals.

\*\*\*\*\*  
NOTE: Where suspended solids are present, specify filters or separators on flush water line. See paragraph entitled, "Centrifugal Abrasives-Separators," in this section.  
\*\*\*\*\*

Bypass flushing water supply shall be free of iron rust products and other

abrasive materials and shall be directed onto face of seal without dead ending. All piping and accessories shall be provided.

Throttling bushing shall have clearances to minimize leakage in case of complete seal failure without restriction of flushing water.

Mechanical seals shall not be subjected to hydrostatic test pressures in excess of the manufacturer's recommendations.

Mechanical-seal manufacturer's representative shall direct on-site seal installation, testing, adjustment, and placing-into-service operations and shall instruct facility personnel as scheduled by the Contracting Officer.

#### 2.1.10 Centrifugal Abrasive-Separators

\*\*\*\*\*  
NOTE: Select the following for cleansing of  
dirt-laden water from discharge of pump used to  
flush and cool mechanical or packing seals.

Check manufacturer's catalog for pressures required,  
particle-size removal, and fluid-flow data.

\*\*\*\*\*

Pump seals shall be flushed with pump discharge water cleansed by centrifugal force in a cyclone abrasive-separator. Separator shall be constructed of AISI Type 316 corrosion-resistant steel. Underflow shall be piped to waste.

#### 2.1.11 Bearings and Lubrication

\*\*\*\*\*  
NOTE: Select the following for engineered-quality  
pumps only. Rewrite to require double-row thrust  
bearings for heavy-duty service.

\*\*\*\*\*

Bearings shall be heavy-duty ball or roller type with full provisions for the mechanical and hydraulic radial and thrust loads imposed by any normal service condition. Bearings shall be manufactured from vacuum-degassed or processed-alloy steel. Thrust-bearing endplay shall not exceed 0.127 millimeter 0.005 inch. Thrust bearings shall be secured to the shaft by threaded collar and locknut. Double-row ball or roller bearings shall be self-aligning. Bearings shall have an L-10 rated life of not less than [30,000] [50,000] [80,000] [ ] hours in accordance with ANSI/ABMA 9 or ANSI/ABMA 11. Shop drawings shall bear manufacturer's certification of bearing life.

\*\*\*\*\*  
NOTE: For commercial quality pumps and certain  
engineered-quality pumps, select the following  
paragraph.

\*\*\*\*\*

Bearings shall be heavy-duty ball or roller type with full provisions for the mechanical and hydraulic radial and thrust loads imposed by any normal service condition. Bearings shall be manufactured from vacuum-degassed or processed-alloy steel. Bearings shall have an L-10 rated life of not less than [30,000] [50,000] [80,000] [ ] hours in accordance with ANSI/ABMA 9 or

ANSI/ABMA 11. Shop drawings shall bear manufacturer's certification of bearing life.

\*\*\*\*\*

**NOTE: Normally select grease-lubricated bearings  
for low temperatures to minimize condensation and  
contamination.**

\*\*\*\*\*

[Bearings shall be permanently lubricated sealed bearings]

[Bearings shall be grease lubricated and shall be provided with grease supply and relief fittings located at bottom of bearings.]

[Bearings shall be oil-flood lubricated. Oil sumps shall be fitted with 120 milliliter 4-ounce constant-level sight oilers and positive means of sump drainage and condensate detection. Provisions shall include FM-approved brass cocks connected to lowest part of bearing sump.]

Bearing housings shall be cast iron, self-aligning on metal-to-metal surfaces and shall totally enclose bearings.

#### 2.1.12 Flexible Coupling

Pump shaft shall be connected to the motor shaft through a flexible coupling. Flexible member shall be a tire shape in shear, or a solid-mass serrated-edge disk shape made of chloroprene materials and retained by fixed flanges. Flexible coupling shall act as a dielectric connector and shall not transmit sound, vibration, or end thrust.

[All couplings in intermittent on/off service shall have couplings selected on the basis of a 2.0 service factor. Other service factors shall be in accordance with the manufacturer's instructions.]

[Pumps shall be provided with spacer couplings.]

#### 2.1.13 Bedplate

Pump and driver shall be mounted on a common bedplate, hollow cast iron, multiribbed for maximum rigidity, with adequate number of grout holes and grout air vents, and with drip rim and drain tapping.

Contractor shall submit for approval, when specified, a fabricated steel base constructed of a rolled structural-steel perimeter frame, reinforced and cross-braced internally with pipe or rolled structural members, capped with 6 millimeter 1/4 inch steel plate, and provided with adequate grout holes, grout air vents, drip rim, and drain tapping. Formed or bent steel bedplates are not acceptable.

#### 2.1.14 Motors

Pump motors shall be checked for current direction of rotation only after pumps have been primed and approved by the manufacturer's representative and the Contracting Officer.

Motors shall conform to ANSI/NEMA MG 1

## 2.1.15 Special Requirements

\*\*\*\*\*

NOTE: Specify hereunder any detail, deviation, or waiving of requirement which is pertinent only to specific pumps. Write in headings such as P-1 and P-2, select provided paragraphs, or delete provided paragraphs and write new paragraphs to suit project conditions. For example: pump P-1 shall be fitted with mechanical seals and pump P-2 shall be fitted with packing seals, pump P-4 shall be exempt from vibration testing.

Class ADS: Provide title and specify any detail requirements pertinent only to this class of pumps.

\*\*\*\*\*

Plugged or valved casing drains which may require ip s red-brass pipe shall be brought out beyond periphery of casing to facilitate drainage. Volute plugs at flanges shall be assembled with tetrafluoroethylene tape.

\*\*\*\*\*

NOTE: Class FES or CES: Provide part title and specify hereunder any detail requirements pertinent only to this class and series of pumps.

Class CES pump selections should be limited to 5 horsepower power of 3.75 kilowatt and less. These pumps have special motor shaft requirements. For critical operations where end-suction pumps are desired, use Class FES pumps.

\*\*\*\*\*

Pump casing and pump motor shall be mounted on a single pedestal and shall not require a bedplate. Pump casing drain shall be plugged.

\*\*\*\*\*

NOTE: For very small pumps of one horsepower power of 0.75 kilowatt and less, select either or both of the following two paragraphs after checking manufacturer's literature.

\*\*\*\*\*

Pump casing with threaded inlet and outlet connections shall be furnished.

Pump casing and impeller wear rings shall be furnished.

Pump motor shall be an extended-shaft type with special heavy-duty thrust and radial bearings to accommodate motor and pump thrust loads. Impeller shall be mounted directly on the motor-shaft extension.

\*\*\*\*\*

NOTE: For frequent start/stop or near shutoff head operation and for better extended-shaft support, specify packing.

\*\*\*\*\*

Pump seals shall be packed type.

\*\*\*\*\*  
NOTE: For very small off-the-shelf pumps, select  
the following in lieu of preceding paragraph.  
\*\*\*\*\*

Pump seals shall be packed type or have the manufacturer's standard mechanical seals for the specified service.

## 2.2 BASE-MOUNTED CENTRIFUGAL PUMPS

\*\*\*\*\*  
NOTE: The following is a short-form specification  
with minimum requirements for building trades  
quality, base-mounted pumps. Review project  
requirements and supplement the following as  
necessary.

When using this part, delete paragraph entitled,  
"General Pump Requirements," and subparagraphs.

\*\*\*\*\*

Pumps provided shall conform to ISO 2858 and ISO 5199 HI SCRRP standards for centrifugal pumps and to requirements specified herein.

### 2.2.1 Pump Schedule

Pump capacity design requirements, and characteristics not specified herein shall be as indicated on the pump schedules.

### 2.2.2 Classification

\*\*\*\*\*  
NOTE: Select from the following or include  
description in schedule.  
\*\*\*\*\*

Pump class shall be as scheduled.

Class ADS, axially (horizontally) split-case, single-stage, double-suction, centrifugal type

\*\*\*\*\*  
NOTE: Limit all end-suction pumps to 1,000 gpm at  
100 psi 4.0 cubic meter per minute at 690 kilopascal.  
\*\*\*\*\*

Class FES, radially (vertically) split-case, single-stage, frame-mounted, end-suction centrifugal type

### 2.2.3 Pump Selection

\*\*\*\*\*  
NOTE: Check hydraulic conditions for each  
application with at least three manufacturers to  
assure compliance with the following efficiency  
conditions.  
\*\*\*\*\*

Pumps shall be selected at the point of maximum efficiency for a given

impeller/casing combination. Deviations within 3 percent of maximum efficiency are permissible, provided that the efficiency is not less than the scheduled efficiency. Pumps having impeller diameters larger than 90 percent of the published maximum diameter of the casing or less than 15 percent larger than the published minimum diameter of the casing will be rejected.

\*\*\*\*\*  
NOTE: A centrifugal pump always operates at the intersection of its head-capacity curve and the system curve that shows the head required to make the liquid flow through the system of piping, valves, and fittings.  
\*\*\*\*\*

#### 2.2.4 Balancing

\*\*\*\*\*  
NOTE: Pump and Motor balance shall conform to ISO 1940-1 - (1986) Balance Quality Requirements of Rigid Rotors - Determination of Permissible Residual Unbalance unless otherwise noted. Motor vibration levels shall conform to NEMA Specification MG-1, Motors and Generators, Part 7 unless otherwise noted.  
\*\*\*\*\*

Pump impeller assemblies shall be statically and dynamically balanced to ISO 1940-1-1986, [G6.3] [G2.5] [G1.0] [\_\_\_\_\_] minimum. Correction planes needed for additional mass weight for balancing shall be determined by using a calibrated certified balancing machine capable of identifying the magnitude and angular position of any unbalance of the impeller.

#### 2.2.5 Casing

\*\*\*\*\*  
NOTE: Review casing pressure rating.  
\*\*\*\*\*

Pump casing shall be bronze-fitted cast iron with a design working pressure of not less than 862 kilopascal at 93 degrees C 125 psig at 200 degrees F. Casing piping connections in 50 millimeter 2-inch and larger sizes shall be flanged and shall conform to ASME B16.1 MSS SP-51, MSS SP-86 and ISO 7005-2. Casing shall have trap-equipped openings for air venting, priming, draining, and suction and discharge gages. Pump shall be convertible to packing service without machining of casing.

#### 2.2.6 Wearing Rings

Wearing rings shall be provided in every pump case and on all impellers larger than 200 millimeter 8-inch diameter.

#### 2.2.7 Shaft

[Shaft shall be solid, sleeveless, AISI 400 series corrosion-resistant steel, hardened to 425 Brinell in stuffing-box area or sleeved type with AISI 300 series shaft and AISI 400 series corrosion-resistant steel sleeves hardened to 425 Brinell.]

Shaft vibration at sealing face shall conform to the paragraph entitled,



"Pump Acceptance," when pump is operating against shutoff head.

#### 2.2.8 Mechanical Seals

Mechanical seals shall be the manufacturer's standard for the specified and indicated service. Bypass flushing water supply shall be free of iron rust products and other abrasive materials and shall be directed onto face of seal without dead ending. All piping and accessories necessary to the function shall be provided.

#### 2.2.9 Bearings and Lubrication

Bearings shall be heavy-duty ball or roller type and shall have an L-10 rated life of not less than [30,000] [50,000] [80,000] [\_\_\_\_] hours in accordance with ANSI/ABMA 9 or ANSI/ABMA 11.

\*\*\*\*\*  
NOTE: Select oil-lubricated bearings only for  
water-heating systems.  
\*\*\*\*\*

[Bearings shall be permanently lubricated sealed bearings]

[Bearings shall be grease lubricated unless otherwise specified and shall be provided with grease supply and relief fittings located at bottom of bearing.]

[Bearings for hot-water service shall be oil-flood lubricated. Oil sumps shall be fitted with 120 milliliter 4-ounce constant-level sight oilers and positive means of sump drainage and condensate detection. Provisions shall include FM-approved brass cocks connected to lowest part of bearing sump.]

#### 2.2.10 Flexible Coupling

Pump shaft shall be connected to the motor shaft through an elastomeric flexible member in shear and shall be a tire shape or a solid-mass serrated-edge disk shape retained by fixed flanges. Flexible coupling shall act as a dielectric connector and shall not transmit sound, vibration, or end thrust.

#### 2.2.11 Bedplate

Pump and driver shall be mounted on a common bedplate which shall be constructed for maximum rigidity, with an adequate number of grout holes and grout air vents and with drip rim and drain tapping.

#### 2.3 LINE-MOUNTED PUMPS

\*\*\*\*\*  
NOTE: The following is limited to pumps up to and  
including 1 horsepower 0.75 kilowatt.  
\*\*\*\*\*

Pump shall be the capacity indicated.

Design, construction balancing, and vibration-isolation provisions shall conform to ISO 5199 and ISO 2858 HI SCRRP standards specifically for quiet waterside operation.

Pump casing and seal shall be suitable for operation at pressures up to 862 kilopascal 125 pounds per square inch at temperatures to 121 degrees C 250 degrees F.

Pump casing shall be cast [iron] [bronze].

Pump shall be single-stage, single-suction centrifugal-type, bronze, fitted with mechanical seal. Pump shaft shall be AISI 300 series corrosion-resistant steel or the manufacturer's standard alloy steel with nonferrous-metal-protected wetted surfaces. Coupling shall be elastomer-in-shear or four-spring type. Mechanical seal shall be designed for service with makeup water normal to the site and for 300 ppm of hexavalent chromate in the recirculating system. Pump bearings shall be oil-lubricated sleeve type, with oil reservoirs for extended periods of service without requiring added lubricant.

Motor shall have oil-lubricated sleeve bearings with oil reservoirs for extended periods of service, without requiring added lubricant, and shall be resiliently mounted.

### PART 3 EXECUTION

#### 3.1 PUMP PROTECTION

Before any pump is operated, sumps and piping systems shall be cleaned and flushed to remove all particles larger than 1000 micro meter or larger than one-half of the smallest pump axial or radial clearance, whichever is smaller. Permanent and temporary pipeline strainers shall be in place and shall be cleaned frequently to prevent cavitation. Temporary strainers shall not be removed until after system acceptance, unless otherwise approved.

Mechanical-seal flushing water shall be provided with centrifugal separator or 10-micrometer filter element where loose rust may be present at startup.

#### 3.2 GROUTING

\*\*\*\*\*  
**NOTE: Premature failure of seals and bearings can be caused by the deterioration of machinery bases and foundations. The cause of the deteriorations is often due to loading of the base and/or foundation before the concrete or grout has properly cured.**  
\*\*\*\*\*

Shimming, alignment, and grouting of pump, driver, and bedplate shall be in accordance with the most stringent requirements of the manufacturer's instructions and as specified herein. Grout shall cure a minimum of [28] [\_\_\_\_\_] calendar days before being loaded. After grouting has cured, bedplate shall be hammer-tested for voids. Poor grouting, as evidenced by voids, shall require resetting of pump assembly.

#### 3.3 VIBRATION ISOLATION

\*\*\*\*\*  
**NOTE: Vibration-isolation provisions shall be selected for each application.**

**Mass of inertia block may be up to three times**

weight of supported assembly.

Drawings should show piping isolation, anchorage,  
and pump and pipe support details.

\*\*\*\*\*

Vibration isolation shall conform to the provisions of Section  
23 05 48.00 40 VIBRATION ISOLATION FOR AIR CONDITIONING EQUIPMENT.

### 3.4 ALIGNMENT

Before attempting alignment, the contractor will demonstrate that the pump does not have any load/force imposed by the piping system. Minimum alignment values (below) are for pump and driver at normal running temperatures. Values must be compensated for thermal growth. Limited movement of the pump or driver (commonly known as bolt-bound) must be corrected to ensure alignment capability. Hold down bolts shall not be undercut in order to perform adjustment.

Shims shall be commercially die-cut, without seams or folds, and be made of corrosion resistant stainless steel. No more than four shims shall be used at any single point.

Units with drive motor over [7.5] [10] [15] [20] [25] hp shall have alignment jack bolts installed.

Pump and driver shall be aligned to the following minimum specifications:

Speed (RPM)	close-coupled offset (mils)	close-coupled angle (mils/in.)	spool piece angle (mils/in. @ coupling pt.)
600	6.0	2.0	3.0
900	5.0	1.5	2.0
1200	4.0	1.0	1.5
1800	3.0	0.5	1.0
3600	1.5	0.4	0.5
7200	1.0	0.3	0.4

[Pump alignment shall be performed under the direction of the manufacturer's representative.]

Final alignment settings shall be provided as part of the final test data.

Pump shall be dowelled in place with AISI 18-8 corrosion-resistant steel spiral-wrapped pins before being subjected to pressure or piping reaction. After grouting and final alignment, and no sooner than after 40 hours of continuous operation, the driver shall be similarly dowelled in place. Taper pins are not acceptable.

### 3.5 Vibration Analyzer

Contractor shall use an FFT analyzer to measure vibration levels. It shall have the following characteristics: A dynamic range greater than 70 dB; a minimum of 400 line resolution; a frequency response range of 5Hz-10 KHz (300-600000 cpm); the capacity to perform ensemble averaging, the capability to use a Hanning window; auto-ranging frequency amplitude; a minimum amplitude accuracy over the selected frequency range of plus or minus 20 percent or plus or minus 1.5 dB.

An accelerometer, either stud-mounted or mounted using a rare earth, low mass magnet and sound disk(or finished surface) shall be used with the FFT analyzer to collect data. The mass of the accelerometer and its mounting shall have minimal influence on the frequency response of the system over the selected measurement range.

### 3.6 PUMP ACCEPTANCE

Prior to final acceptance, vibration analysis shall verify pump conformance to specifications. Vibration levels shall not be more than .075 in/sec at 1 times run speed and at pump frequency, and .04 in/sec at other multiples of run speed.

Pump shall be operated and demonstrated to be nonoverloading at any operating point and that the flow capacity is as specified. Final test reports shall be provided to the Contracting Officer. Reports shall have a cover letter/sheet clearly marked with the System name, Date, and the words "Final Test Reports - Forward to the Systems Engineer/Condition Monitoring Office/Predictive Testing Group for inclusion in the Maintenance Database."

-- End of Section --